

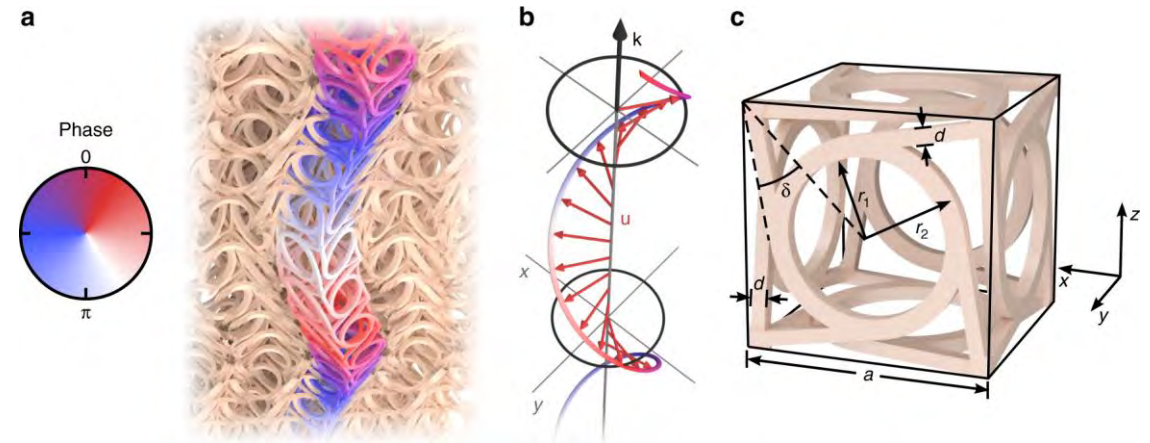
PhD in Mechanical Engineering

Development of novel design methodologies for additive manufacturing

► **Candidate:** Ing. Riccardo Caivano

► **Tutor:** Prof. Giorgio Chiandussi

► **Scholar grant:** Interdepartmental Integrated Additive Manufacturing@PoliTo Centre (IAM) - Design Additive Manufacturing



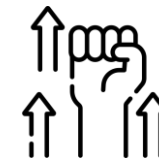
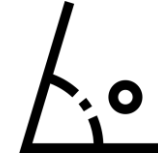
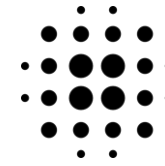
Topology Optimisation for 3D printed composite materials

Problem: Finding the best material distribution and fibre orientation for a composite material

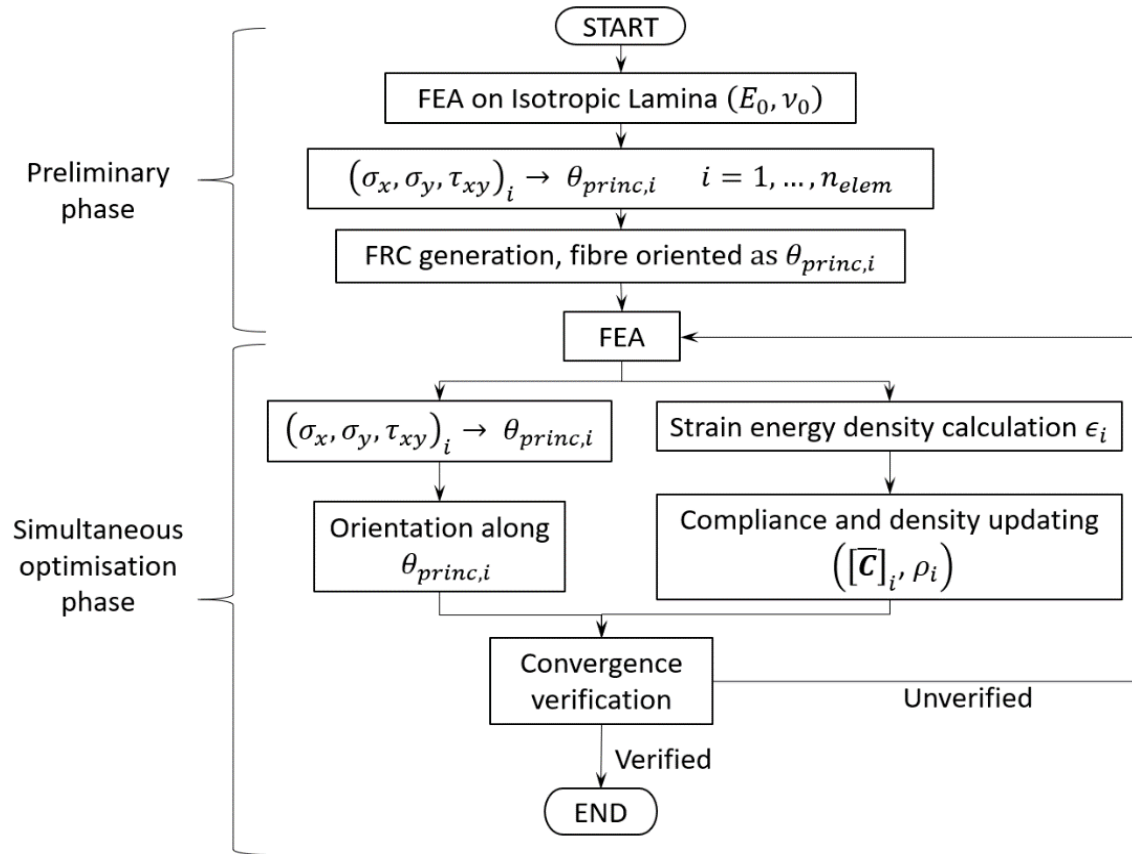
State of art: Many algorithms but computationally expensive and applied on bidimensional cases only

Proposed solution: novel algorithm based on optimality criteria for topology optimisation and tailored fibre orientation suitable for 3D printed composites

Significance: Novel generation of strong, light and fully optimised composite components



► Proposed algorithm solution



Legend

FEA - Finite Element Analysis

E_0, ν_0 - Stiffness and necking coefficient

$(\sigma_x, \sigma_y, \tau_{xy})_i$ - element vector stress

$\theta_{princ,i}$ - element principal stress angle

$[A]$ - stiffness matrix

e_i - element strain vector

j - iteration number

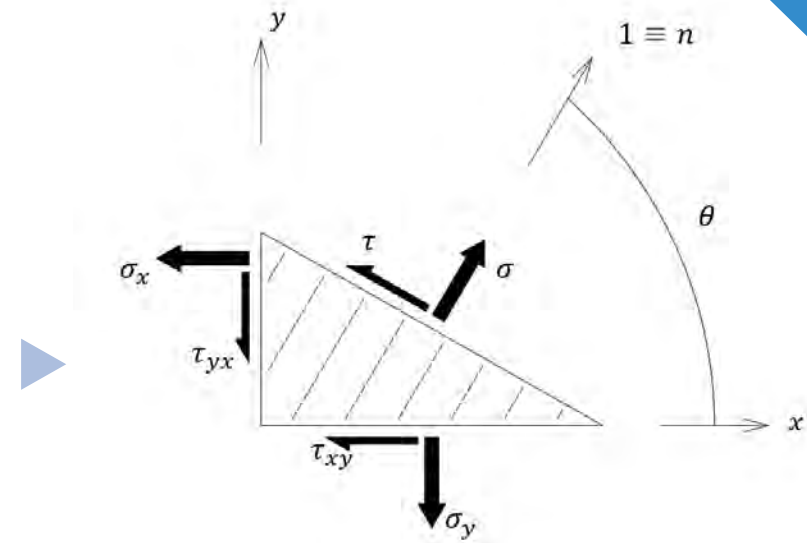
i - element number

FRC - Fibre Reinforced Composite

Ω_i - element volume

ϵ_i - strain energy density

$[\bar{C}]_i$ - element rotated compliance matrix

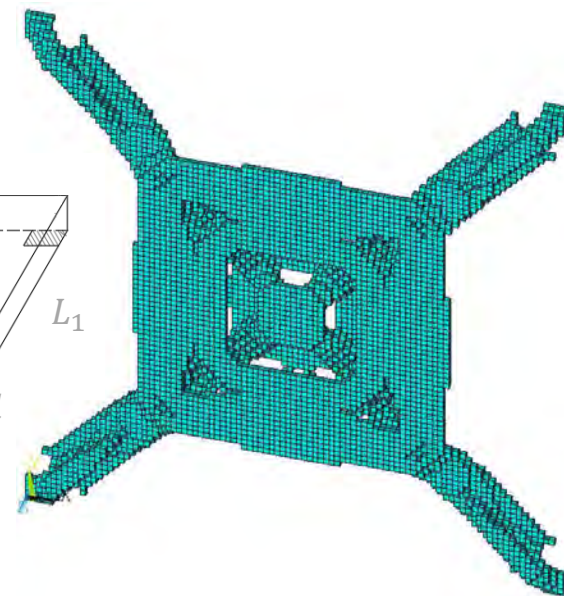
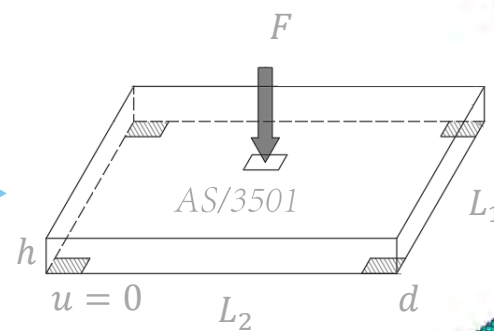
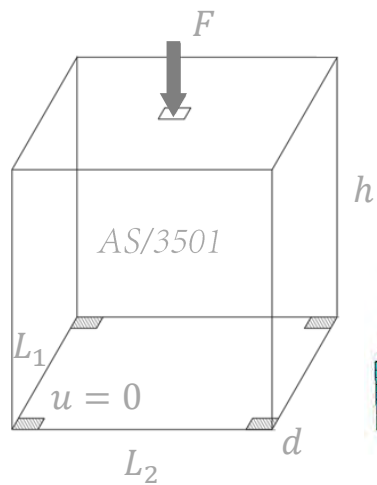
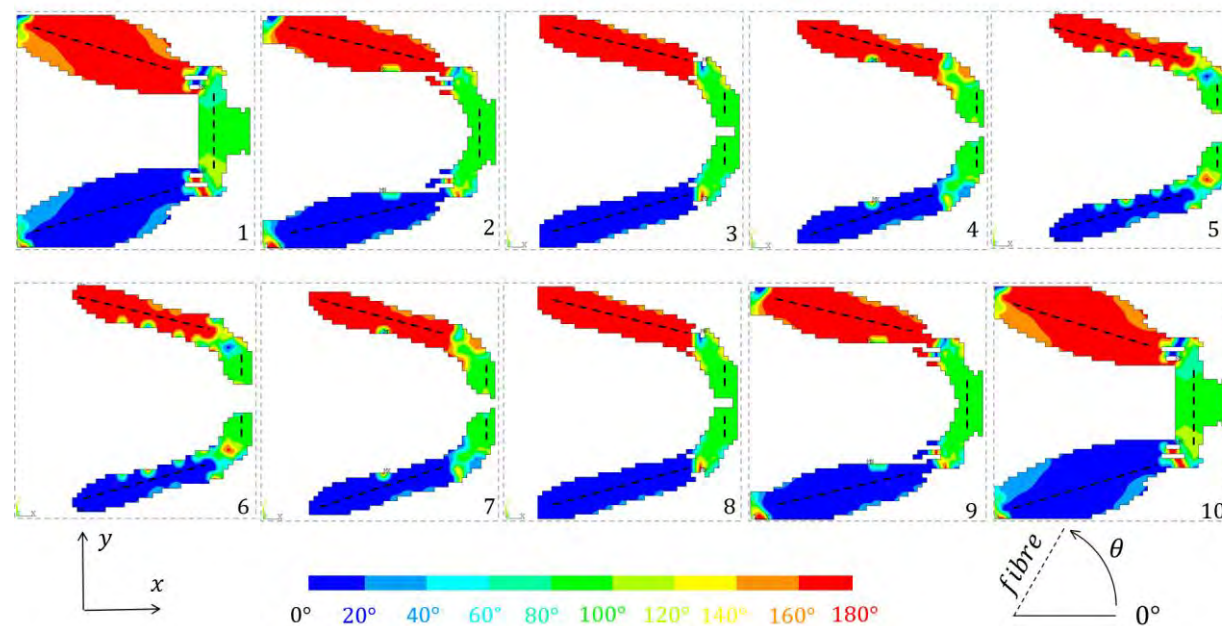
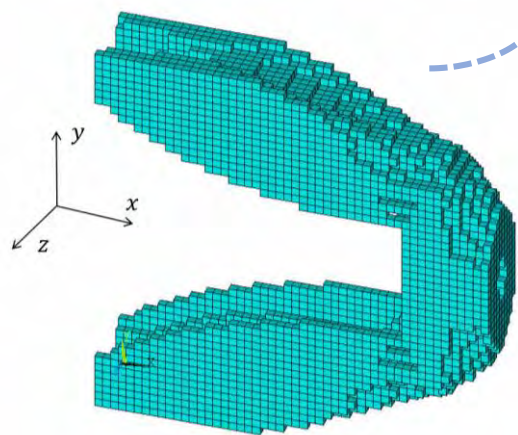
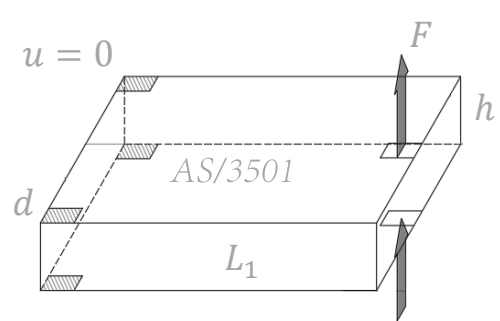


Orientation of fibre reinforcement along principal axes in each element

$$\rightarrow [\bar{C}]_i^{j+1} = \frac{\bar{\epsilon}_i^j}{\epsilon_i^j} [\bar{C}]_i^j \rightarrow \epsilon_i = \frac{\frac{1}{2} e_i^T(u) [A] e_i(u) \Omega_i}{\Omega_i}$$

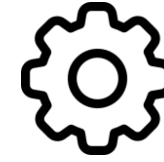
Updating law based on optimality criteria:

Uniform strain energy density



Thermo-structural Topology Optimisation

Problem: Finding the best material distribution to minimise structural compliance and maximise heat exchange concurrently



State of art: Few attempts in thermo-elastic topology optimisation, no coupled constraints



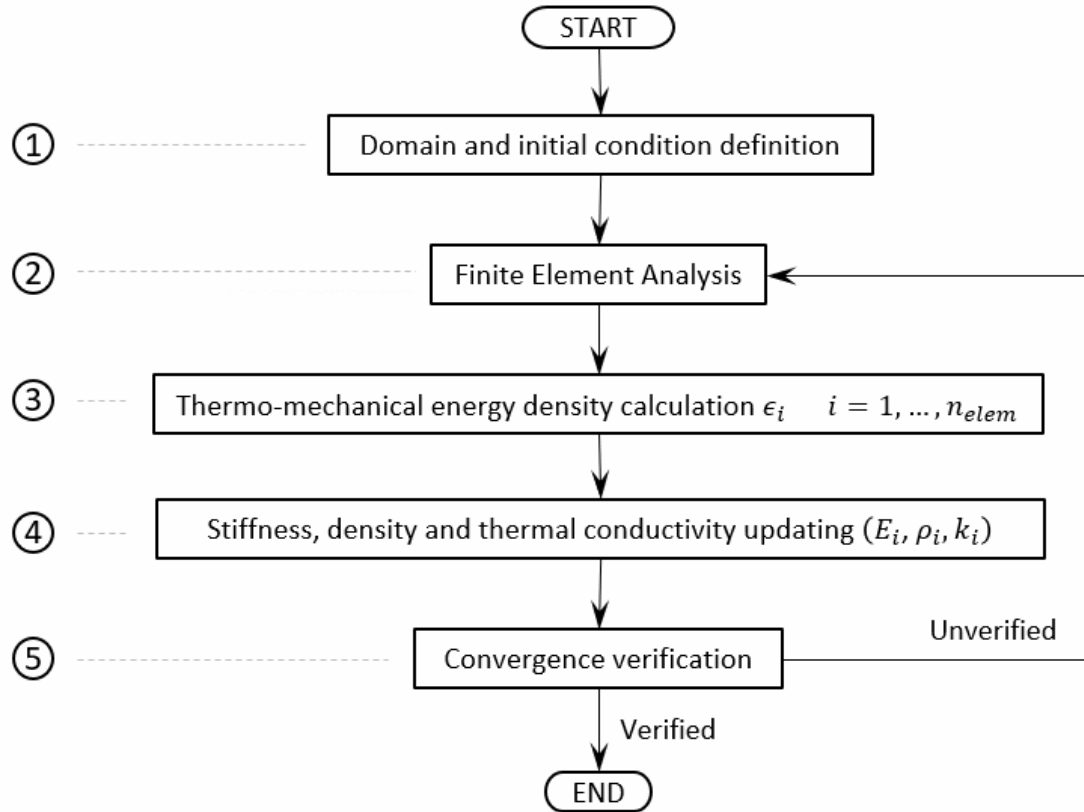
Proposed solution: novel algorithm based on optimality criteria for thermo-structural topology optimisation

Significance: the final optimised component guarantees the maximum stiffness and concurrently the maximum heat exchange, improving lightness and weight saving



R. Caivano, A. Tridello, M. Codegone, G. Chiandussi, *A new methodology for thermo-structural topology optimization: analytical definition and validation.* (Peer Review in progress)

► Proposed algorithm solution



Legend

$\Pi(\mathbf{u}, \theta)$ - thermo-structural potential energy j - iteration number

$\theta_{princ,i}$ - element principal stress angle

$[\mathbf{A}]$ - stiffness matrix

\mathbf{e}_i - element strain vector

ξ - energy linker term

E_i - element stiffness module

i - element number

Ω_i - element volume

k_i - element thermal conductivity

ρ_i - element mass density

ϵ_i - element thermo-structural energy density

Key definition:

Thermo-structural potential energy

$$\Pi(\mathbf{u}, \theta) = -\frac{1}{2} \left(\int_{\Omega} \mathbf{e}^T(\mathbf{u}) \mathbf{A} \mathbf{e}(\mathbf{u}) d\mathbf{x} + \xi \int_{\Omega} k \nabla^2 \theta d\mathbf{x} + \int_{\Omega} \beta \nabla^T \theta \mathbf{u} d\mathbf{x} \right)$$

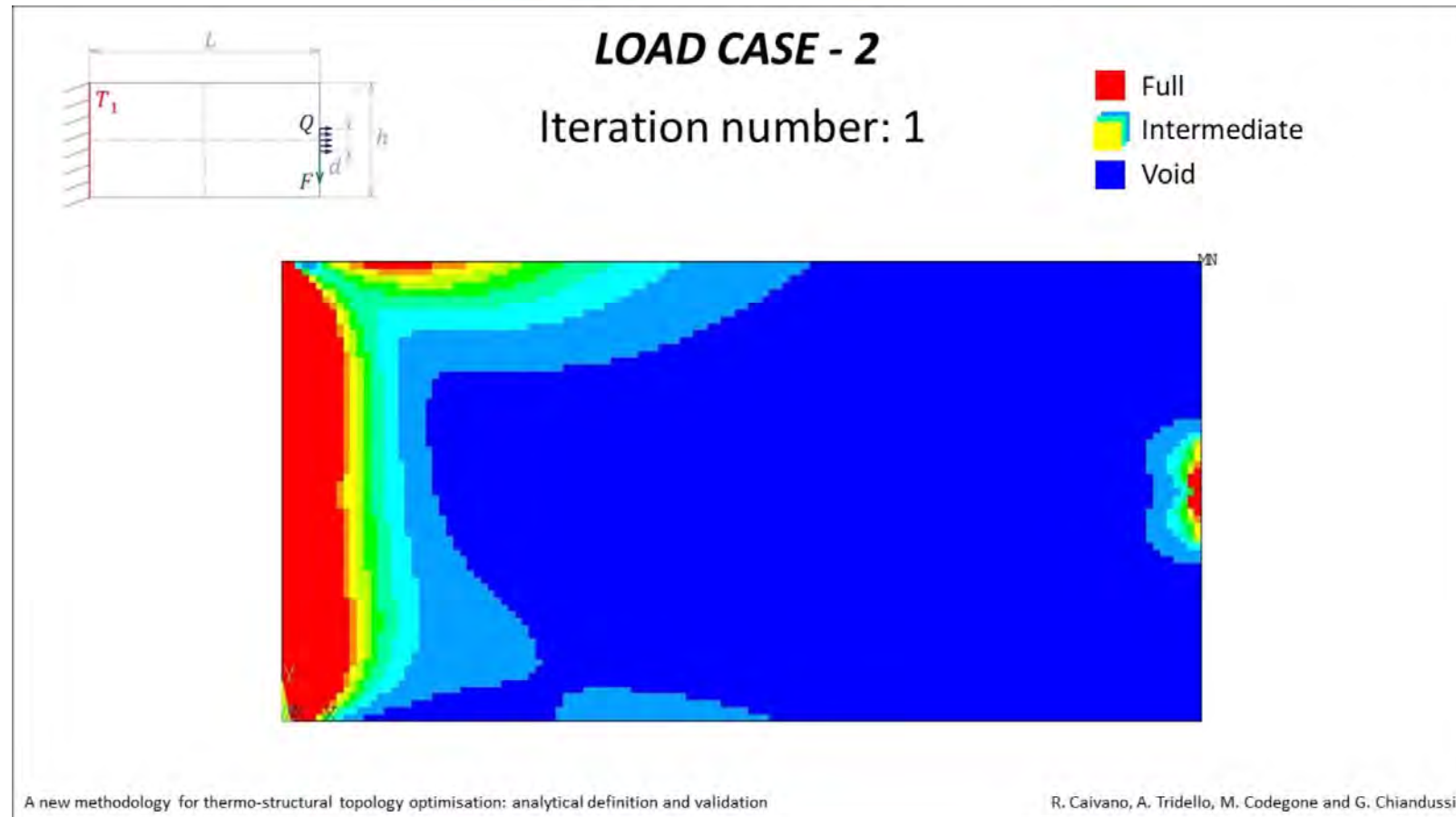
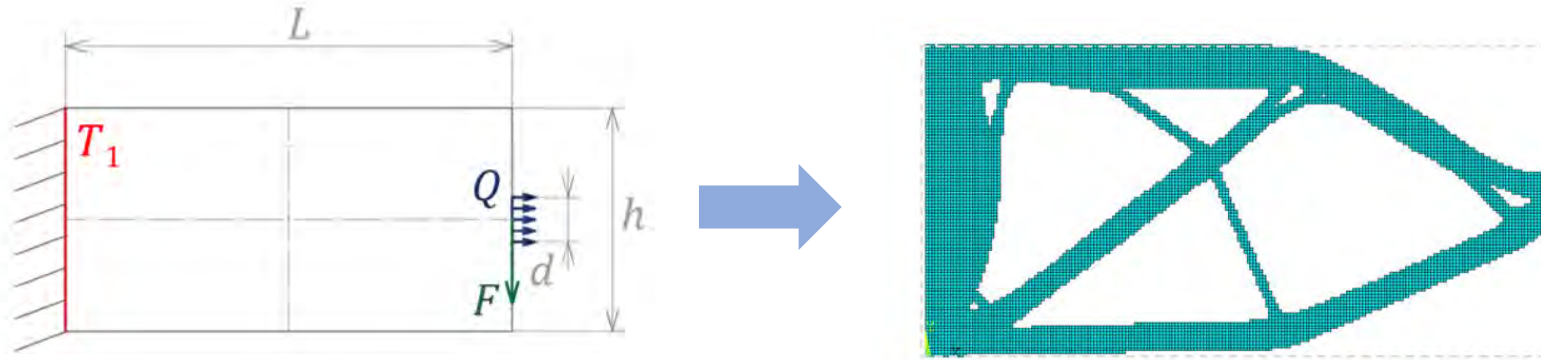
Updating law based on optimality criteria:

Uniform thermo-structural energy density

$$\epsilon_i = \frac{\frac{1}{2} (\mathbf{e}_i^T(\mathbf{u}) \mathbf{A} \mathbf{e}_i(\mathbf{u}) + \xi \nabla_i^T \theta k(x) \nabla_i \theta + \beta \nabla_i^T \theta \mathbf{u}_i) \Omega_i}{\Omega_i}$$

$$\begin{cases} E_i^{j+1} = \frac{\epsilon_i^j}{\epsilon^j} E_i^j \\ \rho_i^{j+1} = \frac{\epsilon_i^j}{\epsilon^j} \rho_i^j \\ k_i^{j+1} = \frac{\epsilon_i^j}{\epsilon^j} k_i^j \end{cases}$$

► Application and test case example



Elasto-static cloaking

Problem: Finding the material distribution which is able to cloak a hole or material variation within a component



State of art: Few attempts with lattice structures and pentamode materials only



Proposed solution: novel algorithm based on numerical optimisation method for isotropic and anisotropic elastostatic cloaking

Significance: the provided material distribution overrides the effect of a hole, what is inside the hole is not affected by external forces, i.e. the perfect case.

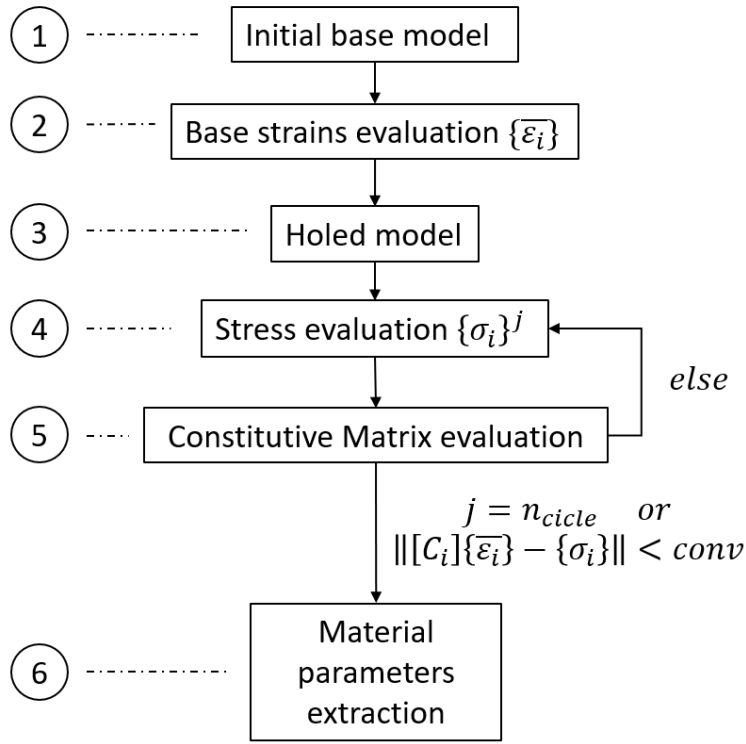


R. Caivano, SS. Injeti, C. Dario,
*A novel numerical approach for elastostatic
cloaking*

In collaboration with Daraio
research group at **Caltech**



► Proposed algorithm solution



Legend

$eig[C_i]$ - eigenvalues of $[C_i]$
 i - element number
 j - iteration number
 $[C_{i,min}]$ - inferior bound
 $[C_{i,MAX}]$ - upper bound
 $[\tilde{C}_e]$ - filtered compliance
 Δ - cloaking error
 $conv$ - convergence parameter

$[C_i]$ - element compliance matrix
 $\{\bar{\epsilon}_i\}$ - reference element strain vector
 $\{\sigma_i\}$ - element stress vector
 $[K_i]$ - global stiffness matrix
 $\{U_i\}$ - global displacement vector
 $\{F_i\}$ - global force vector
 Γ_e - domain external boundaries
 \mathbf{u}_i - element displacement vector
 \mathbf{u}_i^0 - reference for \mathbf{u}_i

Optimisation problem formulation
Resolution by *fmincon* and CVX in MATLAB

$$\begin{cases} \text{Find } [C_i] \\ \min_{[C_i]} \|[C_i]\{\bar{\epsilon}_i\} - \{\sigma_i\}\| \\ \text{s.t.} \begin{cases} [K_i]\{U_i\} = \{F_i\} \\ \forall eig[C_i] > 0 \\ [C_i] \text{ symmetric} \\ [C_{i,min}] \leq [C_{i,j}] \leq [C_{i,MAX}] \\ i = 1, \dots, n_{elem} \end{cases} \end{cases}$$

► Constant filter application

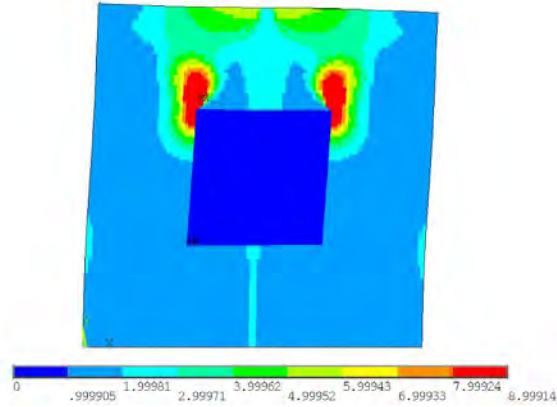
$$[\tilde{C}_e] = \frac{\sum_{i \in \Omega_e} [C_i]}{\sum_{i \in \Omega_e} i}$$

► Cloaking effectiveness measure

$$\Delta = \frac{\sqrt{\sum_{i \in \Gamma_e} (\mathbf{u}_i - \mathbf{u}_i^0)^2}}{\sqrt{\sum_{i \in \Gamma_e} (\mathbf{u}_i^0)^2}}$$

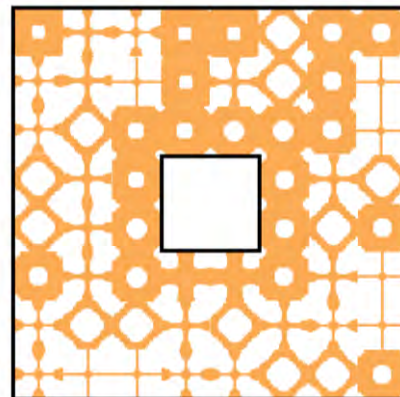
$\Delta \cong 5\%$

Isotropic material

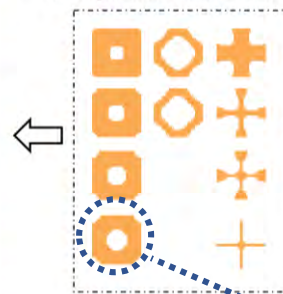


Is there any hole?

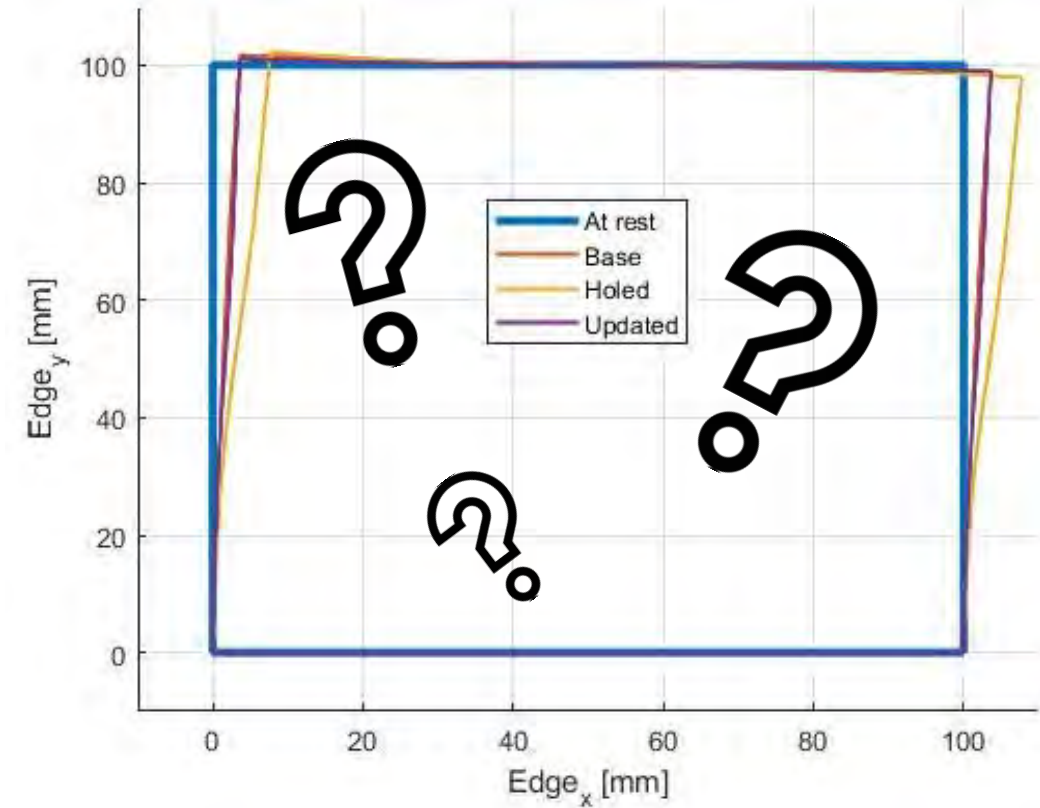
Anisotropic material and Metamaterial application



Set of possible metamaterial units



$\Delta \cong 1,7\%$



$$[C_i] = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ \cdots & C_{22} & C_{23} \\ Sym & \cdots & C_{33} \end{bmatrix}$$

Defect Driven Topology Optimisation

Problem: Finding the best material distribution to minimise structural compliance with stress constraint and fatigue defect driven constraint



State of art: No previous research focused on defect presence in topology optimisation, fatigue considered with classical theory only



Proposed solution: novel algorithm based on numerical optimisation for topology optimisation including the presence of defects

Significance: the final optimised component avoid static failure and fatigue failure thanks to the non-propagability of the most critical defect



R. Caivano, A. Tridello, D. Paolino, G. Chiandussi, F. Berto, X. Gao, Haitao Ma,
Minimum compliance topology optimization of continuum structures with defect driven constraints

In collaboration with NTNU and
Guangdong University of
Technology



 **NTNU**
Norwegian University of
Science and Technology

Topology optimisation formulation

$$\text{find } \boldsymbol{\rho} = \{\rho_1, \rho_2, \dots, \rho_{N_e}\}$$

$$\min_{\boldsymbol{\rho}} C = \mathbf{F}^T \mathbf{U} = \mathbf{U}^T \mathbf{K} \mathbf{U}$$

$$s.t. \begin{cases} \mathbf{K} \mathbf{U} = \mathbf{F} \\ V(\boldsymbol{\rho}) \leq \bar{V} \\ \sigma_e^{\text{VM}}(\boldsymbol{\rho}) \leq \bar{\sigma}^s \\ \sigma_e^1(\boldsymbol{\rho}) \leq \bar{\sigma}^f \quad \text{if } \sigma_1(\boldsymbol{\rho}) \geq 0 \\ \underline{\rho} \leq \rho_e \leq 1 \quad e = 1, 2, \dots, N_e \end{cases}$$

- Density based topology optimisation, SIMP approach
- SIMP issues → continuation method, linear filter, non-linear projection
- Stress constraints → pq -relaxation, K-S aggregation and STM based correction
- MMA solver for the numerical optimisation

$\boldsymbol{\rho}$ - density variable vector
 C - compliance
 \mathbf{K} - global stiffness matrix
 \mathbf{U} - global displacement vector
 \mathbf{F} - global force vector
 N_e - number of elements
 R - stress ratio

Murakami fatigue approach

$$\bar{\sigma}^f = \frac{C_1 \cdot (HV + 120)}{\left(\mu_{\sqrt{a}} + \sigma_{\sqrt{a}} \cdot (-\ln(-\ln(F)) + \ln(V/V_0)) \right)^{\frac{1}{6}}} \cdot \left(\frac{1-R}{2} \right)^{0.226+HV \cdot 10^{-4}}$$

- Defect population lower the fatigue strength
- LEVD defect distribution is supposed
- The most critical defect must not propagate
- 1st principal stress defect-driven constraint

Legend

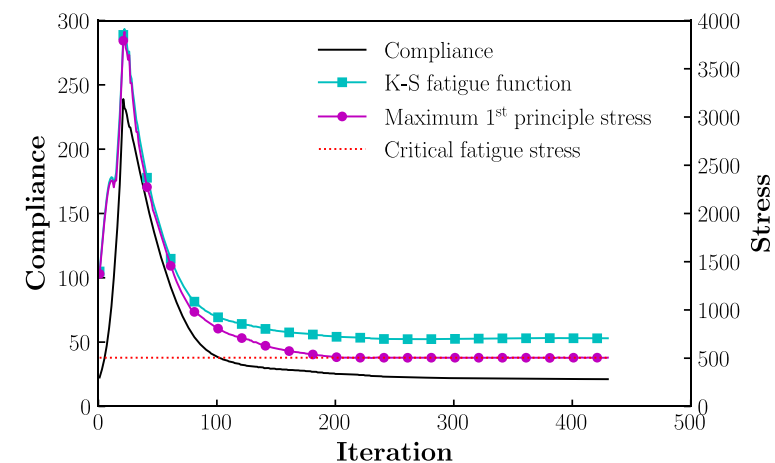
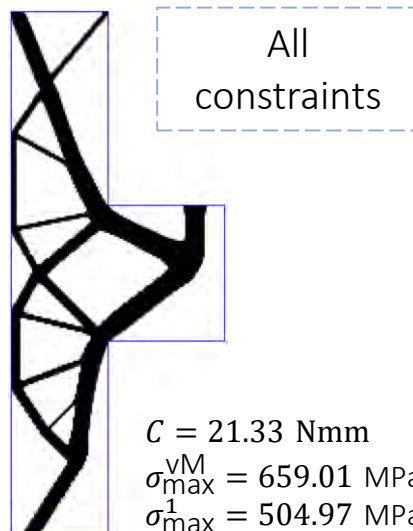
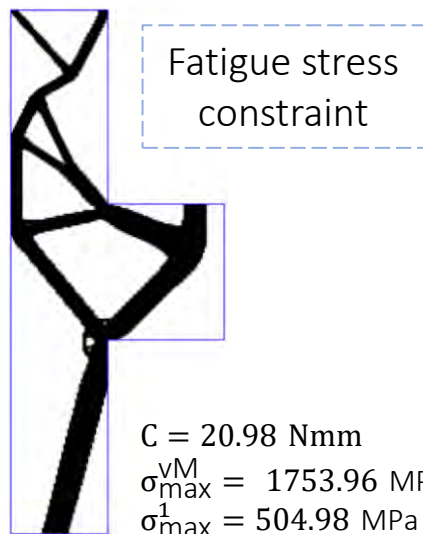
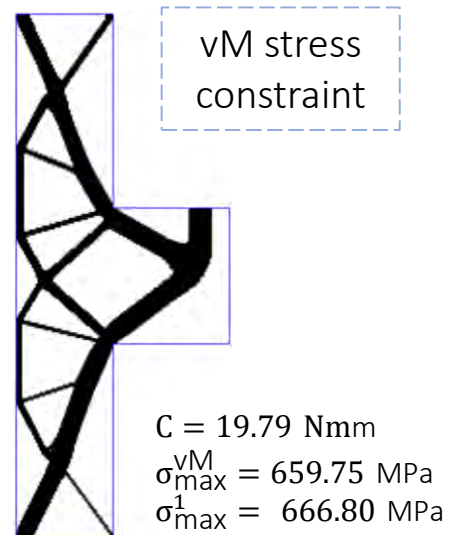
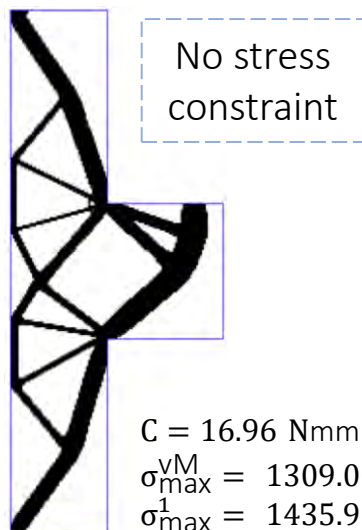
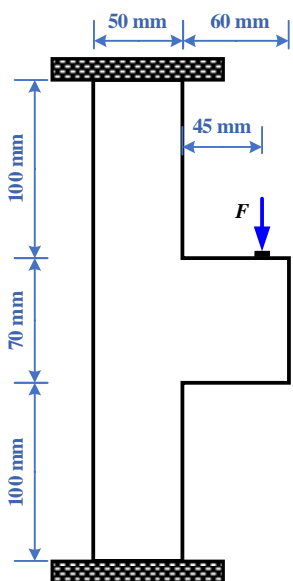
V - volume
 \bar{V} - volume constraint
 σ_e^{VM} - element von mises stress
 $\bar{\sigma}^s$ - max admissible vM stress
 σ_e^1 - element 1st principal stress

C_1 - defect location parameter
 HV - Vickers hardness
 $\mu_{\sqrt{a}}$ - location LEVD parameter
 $\sigma_{\sqrt{a}}$ - scale LEVD parameter
 F - probability
 V_0 - reference volume

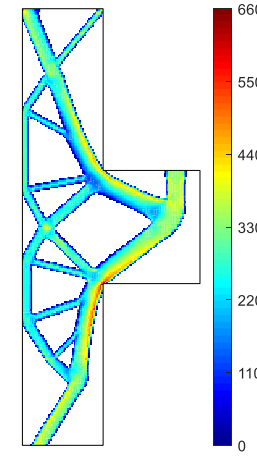
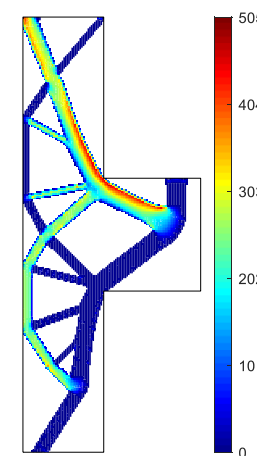
► Application and test case example: Corbel structure with double re-entrant corner

Imposed limits:

$$\begin{cases} \bar{\sigma}^s = 660 \text{ MPa} \\ \bar{\sigma}^f = 505 \text{ MPa} \end{cases}$$



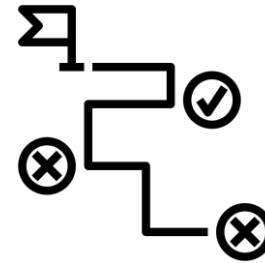
1st principal stress [MPa] Von Mises stress [MPa]



Future outlook for 3rd year PhD

▶ Defect driven topology optimisation:

- ▶ History load case
- ▶ Complex defect population
- ▶ Cracked initial domains



▶ Metamaterial geometry extraction from generic compliance matrix

- ▶ Dynamic Topology Optimisation on 3D composite